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▶ To cite this version:

Jérôme Juillard, Grégory Ardnt, Eric Colinet. Enhancing the frequency stability of a NEMS oscillator with electrostatic and mechanical nonlinearities. Eurosensors'09, Procedia Chemistry, Sep 2009, Lausanne, Switzerland. pp.1415-1418. hal-00445428

HAL Id: hal-00445428 https://centralesupelec.hal.science/hal-00445428

Submitted on 8 Jan 2010

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Enhancing the frequency stability of a NEMS oscillator with electrostatic and mechanical nonlinearities

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Summary

We show how to achieve large-amplitude oscillation and good frequency stability in a nonlinear NEMS oscillator by using a softening nonlinearity to locally balance the hardening behaviour of the Duffing nonlinearity for a given motion amplitude. The case when the softening behaviour is obtained via electrostatic biasing is theoretically investigated, under parallel-plate assumptions, for a fully-capacitive pulse-actuated resonant structure. This is validated by simulated data, in the case of a resonant clamped-clamped beam.

Motivations and results

It is a challenge to achieve large-amplitude motion of NEMS oscillators without deteriorating their frequency stability [1]. On the one hand, achieving large oscillation amplitude leads to better SNR and, thus, simplifies the design of the electronic feedback loop. On the other hand, it results in a nonlinear frequency-amplitude relationship: for example, the frequency instability of a Duffing oscillator is proportional to its oscillation amplitude. In this paper, we show how to achieve large-amplitude oscillation and good frequency stability in a NEMS oscillator: the governing idea is to use a softening nonlinearity to locally balance the hardening behaviour of the Duffing nonlinearity for a given motion amplitude. We study the case when the softening behaviour is obtained via electrostatic biasing. The capacitive detection and actuation scheme is similar to the one in [2] (Fig. 1-2). The system is actuated with short voltage pulses, delivered when the position of the resonant element (typically a clamped-clamped beam) crosses zero: such a feedback scheme ensures that the system oscillates and is stable, even beyond the critical Duffing amplitude [3]. Considering the simpler case of parallel-plate actuation, we use describing function analysis to show that the self-oscillation pulsation ω of the closed-loop system is given by (1) (Fig. 3), where ω_0 is the natural pulsation of the resonant structure, A is the oscillation amplitude, normalized with respect to the gap and γ and δ are positive design-dependent coefficients respectively representing the strength of the Duffing hardening and that of the electrostatic softening. From (1), we establish that:

$$\frac{d\omega}{dA} = 0 \Leftrightarrow A \stackrel{\scriptscriptstyle \Delta}{=} A_{opt} = \sqrt{1 - \left(\frac{\delta}{2\gamma}\right)^{2/5}} .$$
 (2)

Thus, provided the structure is designed so that $\delta < 2\gamma$, there exists an optimal oscillation amplitude around which small perturbations on A have little impact on the value of ω . In order to ensure a large SNR, A_{out} should be made as large as possible, but safely maintained below A_{pi} , for which electrostatic pull-in [5] occurs (Fig. 3-4). Similar analyses can be made in the case of flexible structures; although they are not tractable analytically, they lead to results that are qualitatively similar to (1-2) (Fig. 5). These theoretical results are compared with simulations, in the case of the resonant structure developed in the ANR-funded M&NEMS project (Fig. 6). This design approach remains qualitatively valid for several other actuation schemes (PLL, etc.).

Word count: 496

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